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Date: March 4, 1998

Docket No.: 2008-108P

Assistant Commissioner for Patents  
Washington, DC 20231

Sir:

As authorized by the inventor(s), transmitted herewith for filing is a Reissue application for and on behalf of the inventor(s) according to the provisions of 37 C.F.R. § 1.171.

Inventor(s): Ernest J. Claxton, III  
Eugene F. Poser

For: FIELD-SERVICEABLE SOLIDS-HANDLING VERTICAL TURBINE PUMP

Enclosed are:

X A cut-up copy of U.S. Patent No. 5,496,150 consisting of Eleven (11) pages

X Six (6) sheets of formal drawings

     Certified copy of \_\_\_\_\_

X Executed Declaration (     Original X Photocopy)

     A verified statement (     Original      Photocopy) to establish small entity status under 37 C.F.R. § 1.19 and 37 C.F.R. § 1.27.

X Information Disclosure Statement and Nine (9) references.

     Information Sheet

X Assent of Assignee to Reissue; Establishment of Ownership under 37 C.F.R. § 3.73(b)

X Reissue Oath (Combined with Declaration and Power of Attorney)

The filing fee has been calculated as shown below:

			LARGE ENTITY	SMALL ENTITY
BASIC FEE			\$790.00	\$395.00
	NUMBER FILED	NUMBER EXTRA	RATE FEE	RATE FEE
TOTAL CLAIMS	24 - 20 =	4	x 22 = \$88.00	x 11 = \$
INDEPENDENT CLAIMS	4 - 3 =	1	x 82 = \$82.00	x 41 = \$
MULTIPLE DEPENDENT CLAIMS PRESENTED			+ \$270.00	+ \$135.00
TOTAL			\$960.00	

— The application transmitted herewith is filed in accordance with 37 C.F.R. § 1.41(c). The undersigned has been authorized by the inventor(s) to file the present application. The original duly executed Declaration together with a surcharge will be forwarded in due course.

X A check in the amount of \$ 960.00 to cover the filing fee is enclosed.

— Please charge Deposit Account No. 02-2448 in the amount of \$ \_\_\_\_\_. A triplicate copy of this transmittal form is enclosed.

X Send correspondence to:

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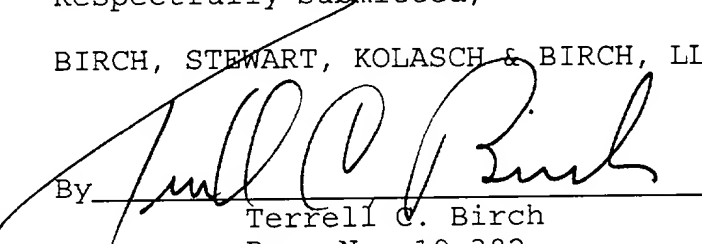
Reissue of Application No. 08/321,857

If necessary, the Commissioner is hereby authorized in this, concurrent, and future replies, to charge payment or credit any overpayment to Deposit Account No. 02-2448 for any additional fees required under 37 C.F.R. §§§1.16, 1.17 or 1.19; particularly, extension of time fees.

Respectfully submitted,

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(Rev. 12-31-97)

# United States Patent [19]

Claxton, III et al.

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[54] **FIELD-SERVICEABLE SOLIDS-HANDLING  
VERTICAL TURBINE PUMP**

[75] Inventors: Ernest J. Claxton, III, Cornelia;  
Eugene F. Poser, Toccoa, both of Ga.

[73] Assignee: Patterson Pump Co., Toccoa, Ga.

[21] Appl. No. 321,857

[22] Filed Oct. 14, 1994

[51] Int. Cl.<sup>6</sup> ..... F04D 11/00; F04D 29/04

[52] U.S. Cl. .... 415/229; 415/901; 415/219.1

[58] Field of Search ..... 415/210.1, 211.2,  
415/218.1, 219.1, 221, 222, 229, 901; 417/423.12,  
424.1

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[11] **Patent Number:** 5,496,150  
[45] **Date of Patent:** Mar. 5, 1996

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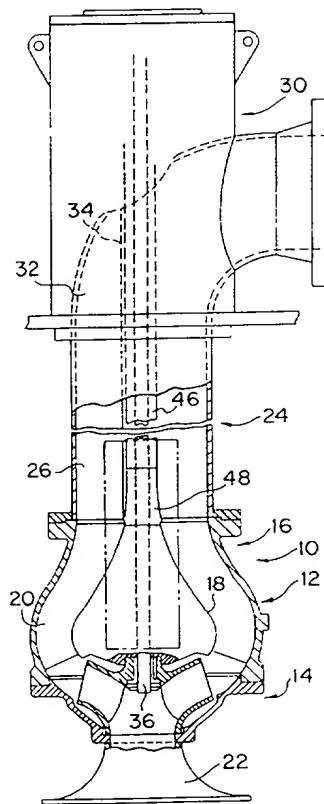
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#### [57] ABSTRACT

A vertical turbine pump which operates with its inlet submerged in a body of liquid to be pumped employs an impeller provided with two or three equiangularly spaced vanes and a diffuser section provided with three equiangularly spaced stationary vanes. A vertical turbine pump incorporating such a combination of impeller and diffuser vanes exhibits favorable hydraulic efficiency and smooth operation and can effectively handle liquids with entrained solids. The pump bowl bearings are part of a cartridge which is readily removed and installed through the upstream end of the pump to facilitate servicing in the field.

**14 Claims, 4 Drawing Sheets**



## FIELD-SERVICEABLE SOLIDS-HANDLING VERTICAL TURBINE PUMP

### BACKGROUND OF THE INVENTION

The present invention relates to a vertical turbine pump and more particularly to such a pump which is resistant to clogging when handling fluids with entrained stringy materials and other solids.

Vertical turbine pumps are well known which operate in an upright position and employ a bowl assembly including a rotary impeller submerged in a body of liquid to be pumped. In these pumps, the impeller develops a diverging tangential flow of the liquid which passes through a bulbous diffusion zone and then through a discharge conduit and elbow.

When a vertical turbine pump is employed for pumping sewage and other liquids having considerable amounts of entrained stringy materials and other solids, clogging of the pump can be a problem. To avoid clogging, careful attention must be given to the size and shape of the passages in the impeller and in the downstream components of the pump. Generally, clog-resistant performance is realized by making the passages as large and as streamlined as possible.

A vertical turbine pump which can effectively handle liquids with entrained stringy materials and other solids is disclosed in U.S. Pat. No. 4,063,849 which issued to Modianos. The Modianos pump incorporates a bowl assembly having a fully shrouded two-vane impeller, a diffusion zone provided with two symmetrically disposed stationary vanes and a discharge column and elbow provided with an axially extending "splitter" vane.

When compared to vertical turbine pumps employing greater numbers of impeller and diffuser vanes, a vertical turbine pump having just two impeller vanes and two diffuser vanes exhibits limited hydraulic efficiency and generates significant pressure pulses which are manifested by relatively rough operation.

Vertical turbine pumps constructed according to known designs employ a pump bowl assembly having bowl bearings which are only accessible after completely disassembling the pump. Because of their location in the lower region of the pump, the bowl bearings are particularly vulnerable to wear and degradation when handling sewage and other corrosive and grit-laden liquids. Of the drive shaft bearings employed in a vertical turbine pump, the bowl bearings are usually the first to require servicing. In known vertical turbine pumps, the disassembly of the pump which is required in order to access the bowl bearings makes servicing of the bearings in the field a difficult prospect.

### SUMMARY OF THE INVENTION

Accordingly, a primary object of the present invention is to provide a vertical turbine pump which can effectively handle solids-laden liquids and also exhibit favorable hydraulic efficiency.

Another object of the present invention is to provide a vertical turbine pump which can handle solids-laden liquids without the pressure pulsations and accompanying rough operation exhibited by vertical turbine pumps employing fewer numbers of impeller and diffuser vanes.

Still another object of the present invention is to provide a solids-handling vertical turbine pump having bowl bearings which can be readily serviced in the field.

The foregoing and other objects of the present invention are realized by a vertical turbine pump comprising: a pump bowl assembly including a casing having a bulbous diffuser section between relatively narrow axially opposed upstream and downstream sections; a rotary impeller disposed in said casing upstream section adjacent a fluid inlet; said impeller incorporating flared shrouds which are axially spaced from each other and spirally oriented impeller vanes disposed between said shrouds and spaced from each other equiangularly about the axis of impeller rotation; a bulbous diffuser core disposed centrally in said casing diffuser section; and three stationary diffuser vanes spaced equiangularly about said diffuser core, said diffuser vanes extending laterally between said diffuser core and said casing and having upstream ends which curve about said diffuser core and downstream ends which extend generally axially through said casing downstream section.

The objects of the invention are also realized by a vertical turbine pump incorporating a pump bowl assembly including a casing having a bulbous diffuser section between relatively narrow axially opposed upstream and downstream sections, a bulbous diffuser core disposed centrally in said casing diffuser section, a drive shaft extending centrally and axially through said diffuser core and a rotary impeller fastened to an end of said drive shaft and disposed in said casing upstream section adjacent a fluid inlet, a bearing cartridge separably fastened within said diffuser core and said bearing cartridge carrying axially spaced bearings which surround and rotatably support said drive shaft.

The detailed description which follows will reveal the further scope of the present invention. However, it should be understood that the detailed description and specific examples are illustrative only, and various changes and modifications within the spirit and scope of the invention may become apparent to persons skilled in the art who have had the benefit of this disclosure.

#### BRIEF DESCRIPTION OF THE DRAWINGS

The accompanying drawings are provided by way of illustration only and should not be regarded as limiting the scope of the present invention.

FIG 1 is an exploded view showing the assembly of the components employed in the vertical turbine pump of the present invention.

FIG 2 is a cross-section in elevation of the assembled pump illustrated in FIG. 1.

FIG. 3 is a cross-section showing details of the diffuser core, bearing cartridge, tubular adapter and drive shaft of the assembled pump illustrated in FIG. 1.

FIG. 4 is a cross-section in elevation of the impeller of the pump illustrated in FIG. 1.

FIG. 5 is a cross-section in plan of the impeller of the pump illustrated in FIG. 1.

FIG 6 is a graph illustrating the performance of a vertical turbine pump constructed according to the teachings of the present invention.

#### DESCRIPTION OF PREFERRED EMBODIMENTS

Referring to FIGS. 1 and 2, the illustrated vertical turbine pump, shown in its upright working position, comprises a bowl assembly 10 including a casing having a bulbous diffuser section 12 located between axially opposed relatively narrow upstream and downstream sections 14 and 16,



respectively. Located centrally within the casing diffuser section is a bulbous, acorn-shaped, diffuser core **18**. Three stationary diffuser vanes **20** extend laterally between the diffuser core and the inner wall of the casing diffuser section. The diffuser vanes are spaced equiangularly about the diffuser core and have upstream ends which curve about the diffuser core and generally axially extending downstream ends. A suction bell **22** is fastened to the casing upstream section.

A discharge conduit **24** fastens to the casing, forming a continuation of the casing upstream section, and incorporates a vane **26** which projects radially into the discharge conduit and extends axially along the direction of flow. This vane is aligned with the downstream end of one of the diffuser vanes and is joined to an interior surface of the discharge conduit. The vane **26** has an interior edge **28** disposed along a central region of the discharge conduit. The discharge conduit may comprise a single conduit or conduit sections which are fastened together. The length of the discharge conduit is chosen to accommodate the particular installation of the pump.

Fastened to the discharge conduit is a discharge elbow **30** incorporating a vane **32** which is aligned with the vane **26** in the discharge conduit. The vane **32** is joined to the interior surface within the outer bend of the discharge elbow and has an interior edge **34** aligned with the interior edge **28** of vane **26** in the discharge conduit.

Incorporated into the discharge elbow are a support for the pump drive motor and a bearing and seal assembly (not shown) for the upper end of the pump drive shaft.

The pump drive shaft **36** extends centrally and axially through the discharge elbow, the discharge conduit and the diffuser core. The lower end of the drive shaft carries an impeller **38** which is disposed beneath the diffuser core within the casing upstream section **14**. As shown in FIGS. 4 and 5, the impeller incorporates three spirally oriented vanes **40** arranged equiangularly about its axis of rotation and between inner and outer upwardly and outwardly flared shrouds **42, 44**.

As best shown in FIG. 2, a tubular shroud **46** extends centrally and axially through the discharge elbow **30** and discharge conduit **24** and encloses the drive shaft. At its upper end the tubular shroud is supported in the discharge elbow. At its lower end, the tubular shroud abuts the upper end of a tubular adapter **48** which engages a surface formed at the top of the diffuser core. An externally threaded connector bearing **50** joins the adjacent interiorly threaded ends of the tubular shroud and the tubular adapter. Within the discharge elbow and discharge conduit, the tubular shroud extends closely adjacent to the interior edges **28, 34** of the vanes **26, 32**, respectively. Intermediate bearings (not shown) may be provided within the tubular shroud to afford added support to the drive shaft if needed. If a tubular shroud made up of sections is used, these bearings may be externally threaded like lower connector bearing **50** in order to serve as couplings for the internally threaded ends of adjacent tubular shroud sections.

As illustrated in FIG. 3, a pair of axially spaced bearings **52, 54** within the diffuser core surround and rotatably support the drive shaft. These bearings are part of a bearing cartridge **56** which comprises a tubular housing **58** surrounding the drive shaft. The bearings, which may be metal or elastomeric sleeves, are fixed within the tubular housing by pressing, for example. The tubular housing fits closely within upper and lower formations **60, 62** provided within the diffuser core. Threads **64** formed on the upper end of the

tubular housing engage with threads 66 formed within the tubular adapter to secure the tubular housing within the diffuser core. To effect axial alignment and prevent relative rotation between the tubular adapter and the diffuser core during installation or removal of the bearing cartridge, mating formations such as circumferentially spaced lugs 68, 70 may be provided on the abutting end surfaces of the tubular adapter and the diffuser core. Other means may be employed for securing the bearing cartridge within the diffuser core. For example, the tubular housing might carry threads which directly engage threads tapped into the upper or lower formations 60, 62 within the diffuser core. Or, the tubular housing might be threaded within an adaptor pressed into the upper formation 60 within the diffuser core, this adaptor could also secure the lower end of the tubular shroud. A retaining ring 70 may be employed in the lower formation of the diffuser core to prevent dislodgement of the tubular housing downwardly through the diffuser core.

To lubricate the bearings in the cartridge, water may be introduced through the bearing and seal assembly at the upper end of the pump drive shaft and into the tubular shroud. Alternatively, oil or grease may be introduced through a bore (or bores) in one or more of the diffuser vanes into the interior of the diffuser core and through a passage (or passages) extending through the sidewall of the tubular housing of the bearing cartridge. To accommodate flow of a viscous lubricant, radial passages may be formed through the bearings in alignment with passages in the tubular housing.

A formation, of circumferentially spaced lugs 59, for example, may be provided at the lowermost end of tubular housing 58 for engagement with a mating formation on the end of a special tool to facilitate turning of the tubular housing during removal or installation of the bearing cartridge.

As best shown in FIG. 4, the lower end of the drive shaft is joined to the impeller by means of a split tapered bushing 72. The taper of a bore within impeller hub 74 matches the externally tapered surface of the bushing. A key 76 fits within the slot in the bushing and keyways formed in the drive shaft and the impeller hub. Bolts (not shown) extend through openings in a flange of the bushing and into the impeller hub. Other openings in the flange may be tapped to accommodate jacking screws which can be used to forcibly separate the bushing and impeller hub. To reduce exposure of the joint between the drive shaft and the impeller hub to the pumped liquid and inhibit corrosion which might impede disassembly of the joint, an elastomeric sealing washer 78 is tightly fitted into the annular space between the end of drive shaft 36 and the counterbore 80 in the impeller hub 74. Preferably, grease is applied to the joint prior to installing the sealing washer to provide an additional barrier between the joint and the pumped liquid.

In a typical installation, the vertical turbine pump of the present invention would be positioned upright with the suction bell disposed below the surface of a body of liquid to be pumped. The rotation of the impeller generates a flow of the liquid upwardly through the passage between diffuser section 12 of the casing and the diffuser core 18, through the discharge conduit and discharge elbow. The diffuser vanes convert the diverging tangential flow from the impeller to an axial flow entering the discharge conduit. By virtue of the three diffuser vanes employed, there is little impediment to flow of solids through the diffuser section. The close spacing of the tubular shroud to the interior edges of the vanes 26, 34 within the discharge conduit and elbow effectively presents a unitary guide vane which prevents stringy materials

from wrapping about the tubular shroud and impeding flow through the pump.

The vertical turbine pump of the present invention having three impeller vanes and three diffuser vanes achieves very favorable hydraulic performance, as shown in FIG. 6 by the head versus flow curves which slope downwardly from left to right and the iso-efficiency curves which are labelled with percentages. (The curves were obtained from testing of a pump having a twelve inch diffuser exit.) This performance is not achieved by a similarly configured vertical turbine pump having two impeller vanes and two diffuser vanes. In addition, the vertical turbine pump of the present invention, configured with two or three impeller vanes and three diffuser vanes, generates less pronounced pressure pulsations and achieves noticeably smoother operation than a vertical turbine pump having two impeller vanes and two diffuser vanes.

Servicing the pump bowl bearings in the field is simple and straight forward. After removing the pump from its working location and supporting the pump horizontally, the suction bell is separated from the pump bowl casing, the impeller is removed from the end of the drive shaft, the retaining ring 70 within the lower formation 62 in the diffuser core is removed and the bearing cartridge is turned to disengage the threads 64 at the upper end of the tubular housing from the threads 66 in the tubular adapter; the bearing cartridge can then be removed through the lower end of the pump bowl casing and replaced with another bearing cartridge having new or reconditioned bearings. The use of a special tool having an end formation which mates with the formation at the lower end of the bearing cartridge facilitates turning of the bearing cartridge during removal and installation.

The invention being thus described, it will be obvious that the same may be varied in many ways. Such variations are not to be regarded as a departure from the spirit and scope of the invention, and all such modifications as would be obvious to one skilled in the art are intended to be included within the scope of the following claims

We claim:

1. A vertical turbine pump comprising:

- a pump bowl assembly including a casing having a bulbous diffuser section between axially opposed upstream and downstream sections which are narrower than said diffuser section;
- a bulbous diffuser core disposed centrally in said casing diffuser section;
- a drive shaft extending centrally and axially through said diffuser core;
- a rotary impeller fastened to an end of said drive shaft and disposed in said casing upstream section adjacent a fluid inlet, said impeller incorporating flared shrouds which are axially spaced from each other and spirally oriented impeller vanes disposed between said shrouds and spaced from each other equiangularly about the axis of impeller rotation,
- three stationary diffuser vanes spaced equiangularly about said diffuser core, said diffuser vanes extending laterally between said diffuser core and said casing and having upstream ends which curve about said diffuser core and downstream ends which extend generally axially through said casing downstream section; and
- a bearing cartridge separably fastened within said diffuser core, said bearing cartridge carrying axially spaced bearings which surround and rotatably support said drive shaft.

2. A vertical turbine pump as recited in claim 1, wherein said impeller incorporates three equiangularly spaced impeller vanes.

3. A vertical turbine pump as recited in claim 1, wherein said impeller incorporates two equiangularly spaced impeller vanes

4. A vertical turbine pump as recited in claim 1, wherein: said bearing cartridge comprises a tubular housing surrounding said drive shaft, and said bearings are fixed within said tubular housing.

5. A vertical turbine pump as recited in claim 4, wherein said tubular housing is fastened within said diffuser core by a threaded coupling.

6. In a vertical turbine pump incorporating a pump bowl assembly including a casing having a bulbous diffuser section between axially opposed upstream and downstream sections which are narrower than said diffuser section, a bulbous diffuser core disposed centrally in said casing diffuser section, a drive shaft extending centrally and axially through said diffuser core and a rotary impeller fastened to an end of said drive shaft and disposed in said casing upstream section adjacent a fluid inlet, the improvement comprising:

a bearing cartridge separably fastened within said diffuser core;

said bearing cartridge carrying axially spaced bearings which surround and rotatably support said drive shaft.

7. In a vertical turbine pump as recited in claim 6, the improvement further comprising:

said bearing cartridge comprising a tubular housing surrounding said drive shaft, and said bearings being fixed within said tubular housing.

8. In a vertical turbine pump as recited in claim 7, the improvement further comprising:

said tubular housing being fastened within said diffuser core by a threaded coupling.

9. In a vertical turbine pump as recited in claim 8, the improvement further comprising:

said threaded coupling comprising threads carried on said tubular housing.

10. In a vertical turbine pump as recited in claim 7, the improvement further comprising:

said tubular housing being fastened to a tubular adapter which surrounds said drive shaft and axially abuts an end surface of said diffuser core which faces downstream.

11. In a vertical turbine pump as recited in claim 10, the improvement further comprising:

said tubular housing being fastened to said tubular adapter by mating threads carried on said tubular adapter and on an end of said tubular housing.

12. In a vertical turbine pump as recited in claim 11, the improvement further comprising:

a formation provided on said end surface of said diffuser core which mates with a formation carried on an end surface of said tubular adapter to effect axial alignment and prevent relative rotation between said tubular adapter and said diffuser core.

13. In a vertical turbine pump as recited in claim 12, the improvement further comprising:

said mating formations comprising axially extending lugs

14. In a vertical turbine pump as recited in claim 7, the improvement further comprising:

a formation provided on an end of said tubular housing adjacent to said impeller, said formation adapted to mate with a tool for facilitating the removal and installation of said tubular housing through said casing upstream section

15. A pump comprising:

a pump assembly including a casing;

a diffuser core disposed in said casing;

a drive shaft extending through said diffuser core;

a rotary impeller fastened to an end of said drive shaft; and

a bearing separably fastened within said casing, said bearing supporting said drive shaft, said bearing being readily removable from an upstream section of said casing.

16. The pump as recited in claim 15, wherein an only active element removed to allow removal of said bearing is said impeller.

17. The pump as recited in claim 15, wherein said bearing is further separably fastenable within said diffuser core.

18. The pump as recited in claim 15, further comprising a bearing cartridge for housing said bearing.

19. The pump as recited in claim 18, wherein said bearing cartridge includes an engagement structure integral therein and said diffuser core includes a reciprocal engagement structure thereon for receiving the engagement structure of said bearing cartridge.

20. An apparatus for facilitating servicing of a bearing in an upstream end of a pump incorporating a pump assembly including a casing, a diffuser core disposed in the casing, a drive shaft extending through the diffuser core, and a rotary impeller fastened to an end of the drive shaft, the apparatus comprising a bearing separably fastened within the casing, said bearing supporting the drive shaft, the bearing being readily removable from an upstream end of the casing.

21. The apparatus as recited in claim 20, wherein an only active element removed to allow removal of said bearing is the impeller.

22. The apparatus as recited in claim 20, wherein said bearing is further separably fastenable within the diffuser core.

23. The pump as recited in claim 20, further comprising a bearing cartridge for housing said bearing.

24. The apparatus as recited in claim 23, wherein said bearing cartridge includes an engagement structure integral therein and said diffuser core includes a reciprocal engagement structure thereon for receiving the engagement structure of said bearing cartridge.

FIG. 1

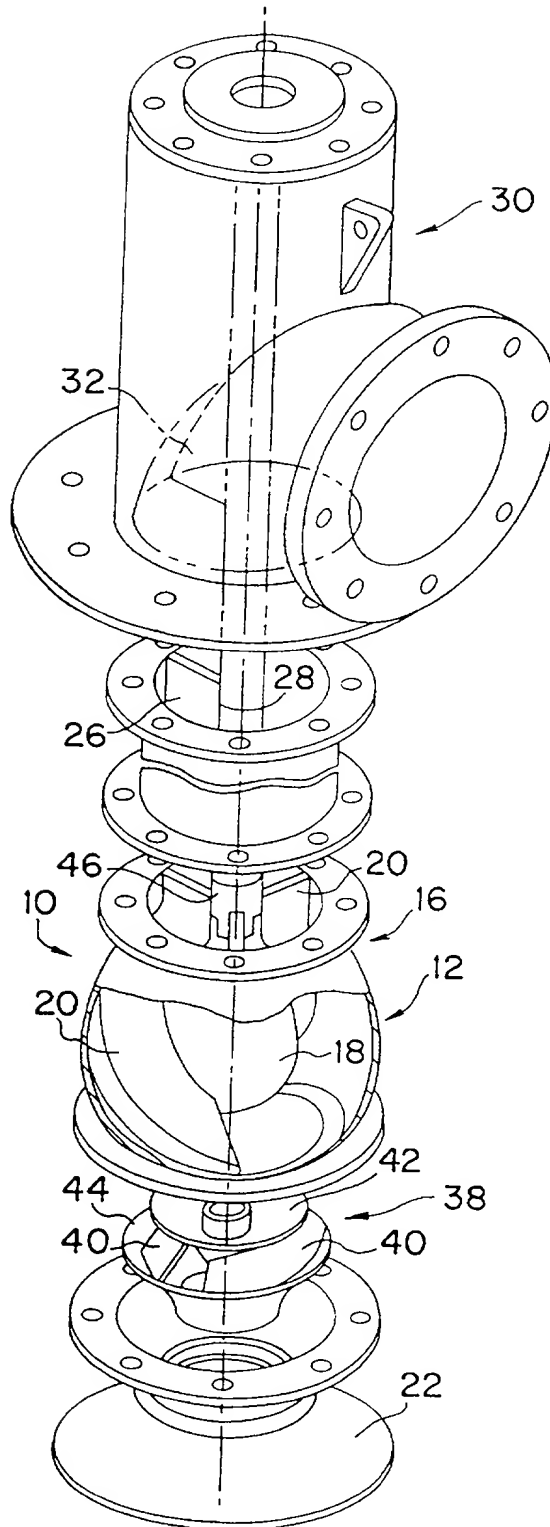
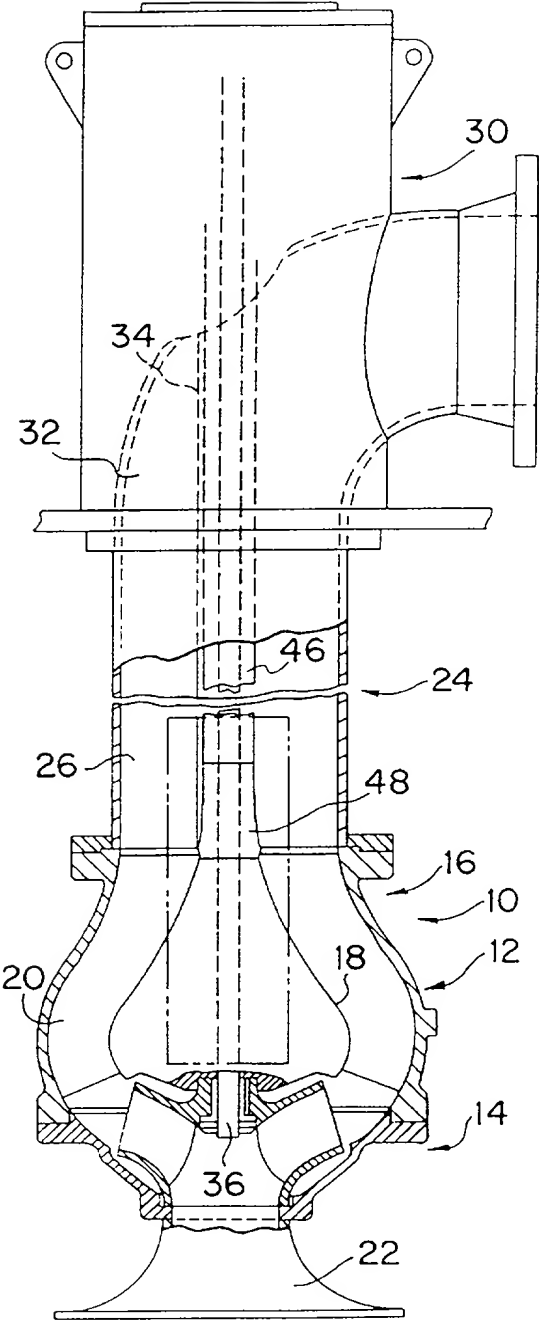


FIG. 2





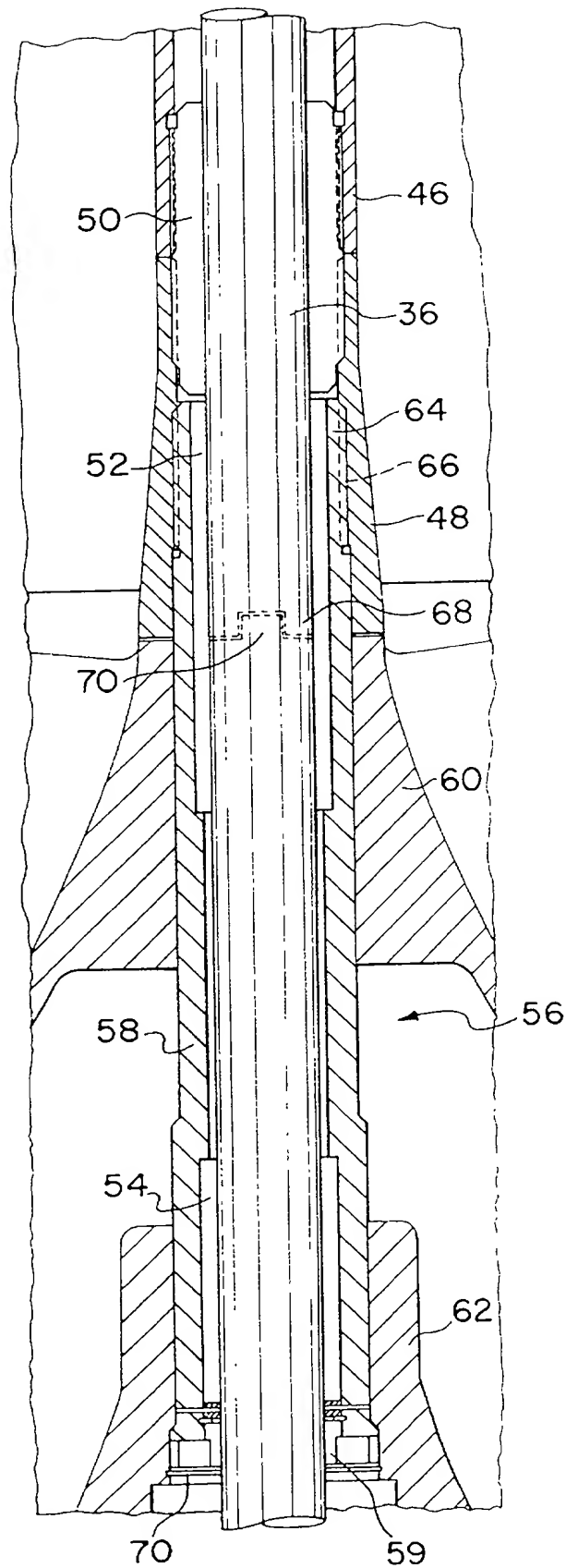
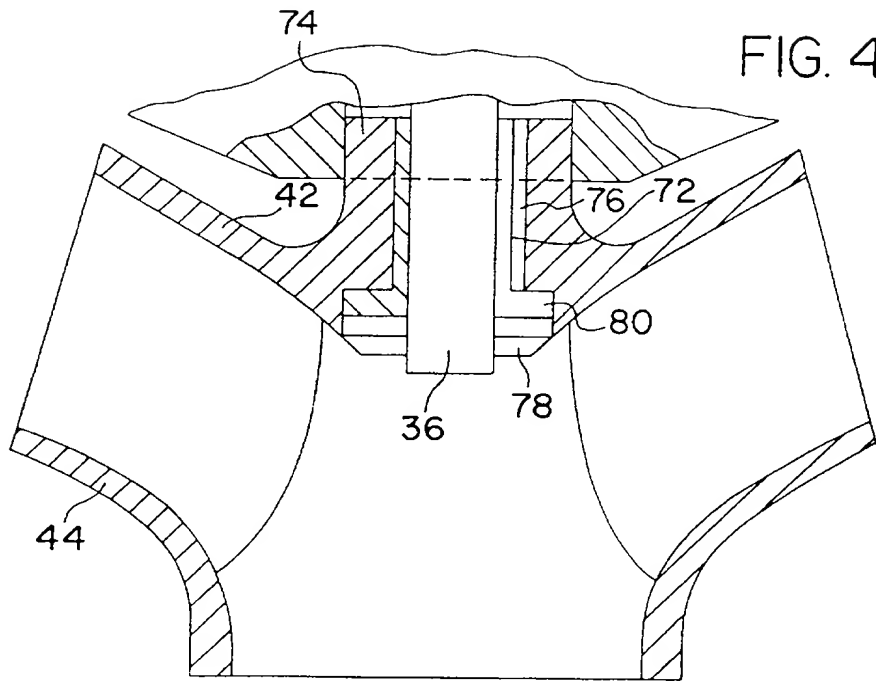


FIG. 3



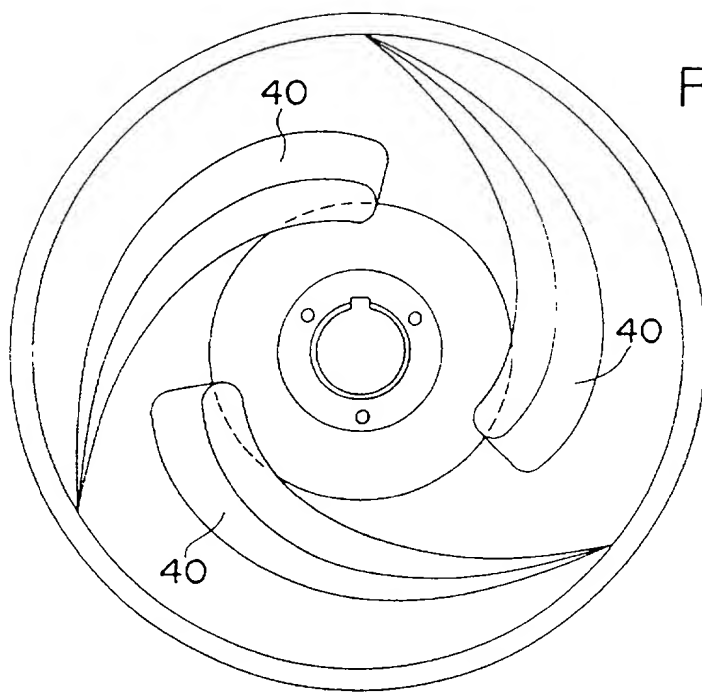
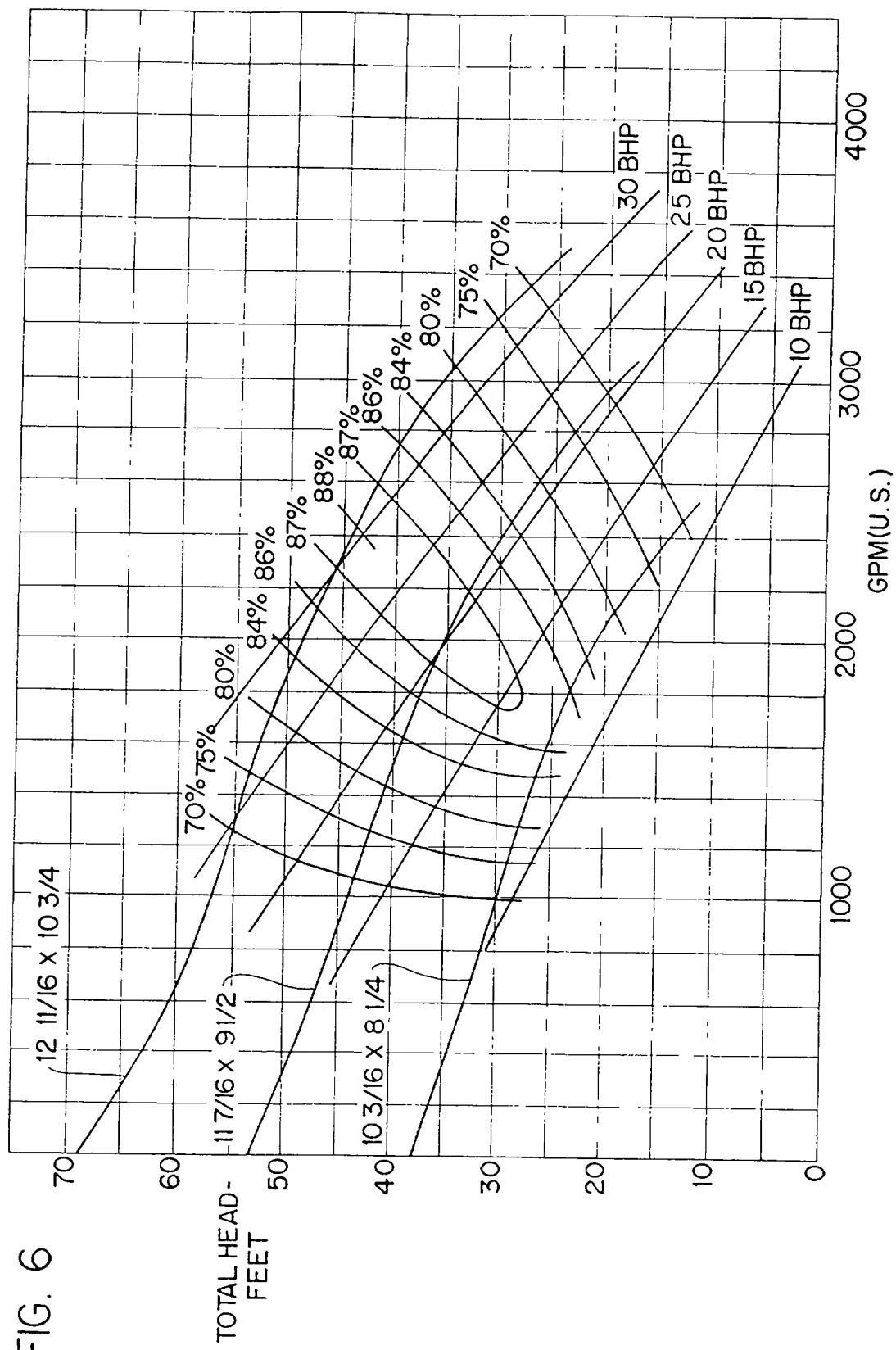


FIG. 5

FIG. 6 is a graph showing the relationship between Total Head (Feet) and Flow Rate (GPM) for various pump configurations. The graph includes curves for different pump sizes (12 11/16 x 10 3/4, 11 7/16 x 9 1/2, 10 3/16 x 8 1/4) and efficiency levels (70%, 75%, 80%, 84%, 86%, 87%, 88%). The Y-axis represents Total Head in Feet (0 to 70), and the X-axis represents Flow Rate in GPM (U.S.) (0 to 4000). The curves show that total head decreases as flow rate increases, and efficiency generally increases with flow rate.

FIG. 6



COMBINED REISSUE DECLARATION AND POWER OF ATTORNEY

As the below named inventors, we hereby declare as follows:

That our names, residence and citizenship are as indicated below;

That we have reviewed and understand the contents of the attached reissue application including original names 1-14 and the newly submitted claims 15-24;

That we acknowledge the duty to disclose information which is material to the examination of this application in accordance with Title 37, Code of Federal Regulations, Section 1.56(a);

That we verily believe that we are the original, first and only joint inventors of the invention described and claimed in United States Patent No. 5,496,150 entitled "Field-Serviceable Solids-Handling Vertical Turbine Pump" and in the foregoing specification for which invention we respectfully solicit a reissue patent;

That we do not know and do not believe that the same invention was ever known or used before our invention or discovery thereof; or patented or described in any printed publication in any country before our invention or discovery thereof, or more than one (1) year prior to the filing of our original application for United States Letters Patent No. 5,496,150 of which that is an application for reissue; or in public use or on sale in the United States of America for more than one (1) year prior to the filing of the original application; that the invention has been patented or made the subject of an inventor's certificate issued before the date of the original application in any country foreign to the United States of America on an application filed by me or my legal representatives or assignees more than twelve (12) months prior to said original application and that no

application for patent or inventor's certificate have been filed by me or my legal representatives or assignees in any country foreign to the United States of America before the application of the original patent;

That we verily believe that there are errors in the original patent which make such original patent partially inoperative by reason of claiming less than we had a right to claim and that such errors occurred without any deceptive intent;

That the original application was directed to a vertical turbine pump having numerous components thereof. That the vertical turbine pump was in part concerned with effectively handling solids-laden liquids and exhibiting favorable hydraulic efficiency. That the vertical turbine pump was also in part concerned with eliminating pressure pulsations and accompanying rough operation. That the vertical turbine pump of the original application was also in part concerned with being readily serviced in the field. To realize all of these objectives, the vertical turbine pump of the invention includes a plurality of diffuser vanes, specific impeller vanes and a bearing separably fastenable within the diffuser core.

That while we originally recognized the importance of each of these aspects of the invention, we did not understand the importance of claiming and thus, when the original application was prepared, we failed to recognize that not all of the details required for realizing all of the aspects were needed and thus, we failed to recognize that the more basic concepts of the invention disclosed in the specification were not covered by the original claims.

That this lack of adequately claiming the invention was due in part to the numerous structural features that were part of the preferred embodiment of our particular pump, without considering how to broadly recite a particular aspect of our pump.

We communicated directly with the attorneys in preparing the application. We had a preliminary search conducted on this application prior to the preparation thereof. This search was concerned with all features of the vertical turbine pump of the invention. We did not advise the U.S. attorneys, and accordingly, they did not fully recognize, that varying levels of importance of each of the aspects of the invention. That we, while realizing that the various descriptions of the pump bowl assembly, the diffuser, the drive shaft and the rotary impeller were those of a preferred embodiment of our invention, and that the removable bearing was useful for improving serviceability of many types of pumps, did not realize the importance of claiming less significant aspects of our invention in a general sense to provide us with the broadest possible protection of our invention to which we are entitled. That we, while recognizing the relative significance of each of the aspects of the invention, did not understand the importance of claiming and thus, we did not realize that we had claimed less than we were entitled to.

That when we executed the Declaration of the original application, we reviewed the application carefully for accuracy, but did not recognize the importance of broadly presently other less significant aspects of the invention and the claims or that individual aspects could be claimed alone. That it was not until after the original Letters Patent

issued that we discovered the importance of claims and discovered that the original presented claims did not adequately define our invention.

That for this reason, there was an error in the original patent claims which rendered the original patent partially inoperative by failure to adequately claim these aspects of our invention;

That, more specifically, with respect to claim 1, the errors include specific recitation of the details of the pump bowl assembly, the diffuser core, the rotary impeller, the three stationary diffuser vanes and the bearing cartridge. That, more specifically, with respect to claim 6, the errors include specific recitation of the axially spaced bearings within a bearing cartridge which surround and rotatably support the drive shaft.

That with respect to claims 1 and 6, new independent claims 15 and 20, respectively, have been presented either to resolve the errors described above by presenting claims which are directed to a pump including broadly claimed pump assembly, diffuser core, rotary impeller and without the recitation of the diffuser vanes, and with the functional relationship of the bearing. The other claims 16-19 depend from claim 15 and recite further details relevant to the aspect regarding which claim 15 is narrower than the original claim 1 and claims 21-24 present similar limitations as to which claim 20 is narrower in that respect to our original claim 7 of the original application.

In summary, claims 1-14 are inadequate to protect our invention as these claims do not encompass the more basic concept of our readily removable bearing cartridge now recited in independent claims 15 and 20. This inadequacy of claims 1-14 requires the



addition of claims 15 and 20. Claim 15 is specifically directed to a pump including a pump assembly having a casing, a diffuser core disposed in the casing, a drive shaft extending through the diffuser core, a rotary impeller fastened to an end of the drive, and a bearing separably fastened within the casing, the bearing supporting the drive shaft, the bearing cartridge being readily removable from an upstream section of the casing. Claim 20 is specifically directed to cover an apparatus for facilitating servicing of a bearing in an upstream end of a pump incorporating a pump assembly including a casing, a diffuser core disposed in the casing, a drive shaft extending through the diffuser core, and a rotary impeller fastened to an end of the drive shaft, the apparatus including a bearing separably fastened within the casing, said bearing supporting the drive shaft, the bearing being readily removable from an upstream end of the casing. Thus claims 15 and 20 have a broader scope than the prior claims regarding the details of the pump and the structure of the bearing within the cartridge that is necessary to cure the inadequacies of claims 1-14 to recite the basic concept regarding the bearing cartridge readily removable from an upstream portion of the casing.

Claims 16-19 and 21-24 are all directly or indirectly dependent from claims 15 and 20, respectively. Each of these claims is necessary to further define the basic element of the invention recited in the independent claims.

Dependent claims 16 and 21 depend on claims 15 and 20, respectively, and further recite that "wherein an only active element removed to allow removal of said bearing is said impeller." This limitation serves to further emphasize how few elements need to be

removed in order to access the bearing, contrary to the conventional downstream access to the bearing. This limitation thus supplements the limitations of the independent claims and is believed necessary to comprehensively protect our invention. Claims 16 and 21 are not equivalent in scope to any of claims 1-14.

Dependent claims 17 and 22 depend on claims 15 and 20, respectively, and further recite that "wherein said bearing is further separably fastenable within said diffuser core." This limitation serves to further emphasize how the bearing is readily removable. This limitation thus supplements the limitations of the independent claims and is believed necessary to comprehensively protect our invention. Claims 17 and 22 are not equivalent in scope to any of claims 1-14.

Dependent claims 18 and 23 depend on claims 15 and 20, respectively, and further recite that "comprising a bearing cartridge for housing said bearing." This limitation serves to further emphasize how the bearing is readily removable by its housing in a bearing cartridge. This limitation thus supplements the limitations of the independent claims and is believed necessary to comprehensively protect our invention. Claims 18 and 23 are not equivalent in scope to any of claims 1-14.

Dependent claims 19 and 24 depend on claims 18 and 23, respectively, and further recite that "said bearing cartridge assembly includes an engagement structure integral therein and said diffuser core includes a reciprocal engagement structure thereon for receiving the engagement structure of said bearing cartridge." This limitation serves to further emphasize how the bearing cartridge is readily removable. This limitation thus

supplements the limitations of the independent claims and is believed necessary to comprehensively protect our invention. Claims 19 and 24 are not equivalent in scope to any of claims 1-14.

Upon review of the prior art cited during the examination of the original application, and the further documents submitted herewith, we do not believe that any of documents disclose or suggest the invention as set forth in any of the claims 1-24, and that we are entitled to the more comprehensive protection offered by the added claims 15-24. As such, we believe that all of claims 15-24 are necessary to protect our invention with claims of varying scope, and to correct for the insufficiencies of claims 1-14.

Applicants hereby offer to surrender the original Letters Patent No. 5,496,150.

Applicants hereby appoint the following as their attorneys, with full power of substitute and revocation, to prosecute this application and transact all business in the United States Patent and Trademark Office in connection therewith, and request that all correspondence with respect to this application be directed to:

BIRCH, STEWART, KOLASCH & BIRCH, LLP  
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Terrell C. Birch	(Reg. No. 19,382)	Raymond C. Stewart	(Reg. No. 21,066)
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Joe McKinney Muncy	(Reg. No. 32,334)	Andrew F. Reish	(Reg. No. 33,443)
C. Joseph Faraci	(Reg. No. 32,350)	Donald J. Daley	(Reg. No. 34,313)

WHEREFORE, the Petitioners hereby offer to surrender, upon the allowance of said application, the original of said Letters Patent and prays that Letters Patent be

2008-108P

reissued to Petitioners for the invention of patent claims 1-14 with the newly presented claims 15-24;

The undersigned declare further that all statements made herein of my own knowledge are true and that all statements made on information and belief are believed to be true; and further that these statements are made with the knowledge that willful false statements and the like so made are punishable by fine or imprisonment, or both, under Section 1001 of Title 18 of the United States Code and that such willful false statements may jeopardize validity of the application or any reissue patent issuing thereon.

Ernest J. Claxton, III

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BIRCH STEWART ET AL

PATTERSON PUMP

FAX NO. 7032058050

P. 29

ASSENT OF ASSIGNEE TO REISSUE PATENT NO. 5,496,150

The undersigned, assignee of the entire interest of U.S. Patent No. 5,496,150 by virtue of an Assignment duly recorded in the Assignment Records of the U.S. Patent and Trademark Office on December 14, 1994 at Reel 7279, Frames 923 - 927, hereby assents to the accompanying reissue application.

PATTERSON PUMP COMPANY

Date: 3-3-98

By:

Albert F. Huber  
President